## PATENT

## **SPECIFICATION**





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## COMPLITE SPECIFICATION

## Improvements in or relating to Needle-roller Bearings

We, Welheld Scharfler and George Schaeffler, both German nationals, trading as Industriewerk Schaeffler olicing the performed, hear Nürnberg. Chrimany, do hereby deciste the invention, for which we pray that a patent may be granted so us, and the method by which it is to be performed, to be particularly described in and by the following statement;—

10 The invention relates to a needle roller bearing of the kind having an annularly grooved inner or outer race, and an axial retaining ring having a closed pliatically deformable dished form which is to find the case of needle bearings having needle cases that such axial retaining rings be construkted not such axial retaining rings be construkted not such axial retaining rings be construkted not seed as divided clastic rings e.g. the known Seger rings, but as closed rings which consists of plastically deformable material and which are pressed with a shape-locking effect into a groove on the outer or inner these rings are given a frusto-comful construction i.e. the upper and lower annular surfaces of the rings form acute agles to the axis of the rings form acute agles to the axis of the rings form acute agles to the axis of the rings form acute agles to the surfaces of the case of rings for insertion into groove i.e. in the case of rings for insertion outer diameter, and in the case of rings to be fitted into grooves formed in outer races, the outer diameter, and in the case of rings to be fitted into grooves formed in line races, in the inner diameter of the ring, is senewhat smaller than the diameter of the outer race, or anator than the diameter of the outer race, or anator than the diameter of the outer race. be fitted into grooves formed in inner races,
if the inner diameter of the ring, is separable than the diameter of the outer race,
or greater than the diameter of the curer race,
or greater than the diameter of the rings are brought into a position corresponding
to the annular receiving groove, grisped by
deforming devices and deformed to constitute a ring having bearing surfaces perpendicular to the axis of the ring so that
after their outer diameter has been inlarged
to rheir inner diameter reduced the rings

benefits futo the grouve and are fixed

penetrate into the growe and are fixed therein.

It is also possible for only one bearing surface of the ring to be of conjust construction at its outer or imper portion, so the that, when the ring is deferenced, the matterial of the annular conjust properties penetrates into the growe and retains the ring therein.

When this type of ring is used difficulties the close tolerances to which the ring must be made. The variations in ring thampter are allowed for, in most cases, by cutting the growes in the inner or outer sace despected than would be necessary for the maximum deformation of the ring affect he ring has been pressed flat. This leads to the following difficulties; if the groove depths are made too large, the rings will not be secured in the radial direction and are therefore capable of turning within the groove. If, on the other hand, the groove depths are too small, the ring bears thou the bottom of the groove producing considerable radial forces becoming greater the shallower the depth of grouve. In this case considerable pressure forces are produced at the prove producing effect of the ring. It is invariably leads to considerable deformation of the groove which, reinforced by the wedging effect of the ring. It invariably leads to considerable deformation of the pressure forces are unsupposed for medical the races are unsupposed for medical to races are unsuppose for medical of the groove in formed. Both loose rings which are free to turn within the grooves and help pressures against the races are unsuppose for medical case of these needle bearings races are less than for races in almost any other general case of application. Therefore, despite the advantages of the closed rings of this type which are constructed without any joint, these rings have not been able to be used up to the present time for retaining in the axial direction needles and cages of needle bearings. cages of needle bearings.

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An object of the invention is to provide a solution which makes it possible to bring a ring of this type into contact with the bottom of the groove after the ring has been pressed into position and thus to secure the ring against rotation, without exerting considerable pressure on the bottom of the groove and thus on the race of the needle bearing.

Dearing.

According to the invention there is provided an axial retaining ring of the kind specified, wherein the dimensions of the ring are such that an inner or outer rim thereof will press against the bottom of the groove, when the ring is fully spread into the groove, which sufficient radial pressure to prevent rotation of the ring in the groove, said rim being shaped so that the aforesaid pressure is transmitted by a surface area of 20 the rim which a only a small fraction of the overall rim the clear to induce part. part

In order that the invention may be clearly understood and readily carried into effect, several emboding at thereof will now be described in defail with reference to the accompanying drawings, to differing scales, in solvich. in which:

Figure 1 is a section showing the outer race of a needle bearing with an axial retaining ring according to the invention in 30 position\_

Figure 2 shows on a somewhat enlarged 35 scale, a peripherally recessed ring in two position prior to insertion in a groove and the second being a suitably deformed position in a groove in the outer race of a needle bearing.

tion in a groove it the outer race of a needle bearing.

40 Figure 3 show in plan and elevation a constructional example of an axial retaining ring having a too thed rim.

Figure 4 is a fection showing the inner race of a needle bearing with an axial retaining ring spread or pressed therein, and Figure 5 shows to a larger scale the construction of a ring as in Figure 4 before insertion within the groove.

Referring now to the drawings, in Figure 50 1 the outer race of a needle bearing is shown at 1. On the inner bearing surface 2 of the outer race travel bearing needles 3 guided in a cage 1. This cage 4, as well as the bearing needles 3, is limited in its axial 55 movement in one direction by an axial retaining ring 5 which is itself secured against axial displacement in a groove 6, formed in the outer race 1 of the bearing. This groove 6 has an oblique outer surface 7 which is 150 sharply inclined to the axis of the bearing. In Figure 2 an outer race 1a of a needle bearing constructed similarly to the race shown in Figure 1, is shown having a groove 6a sharply pointed at 6 and an oblique groove surface 7a. An axial retain-

ing ring 8, having side faces at an oblique angle to the ring axis is provided at its outer rim surface with a preferably central peripheral recess 9 forming two circular edges 10 and 11 of small cross-section. The 70outer diameter of this ring is, in the un-deformed state, of such overall dimensions deformed state, or such overall unissions as to be capable of being inserted into the bors 12 of the outer race 1 without the need for substantial expenditure of force. 75 When the ring 8, which after initial insertion, is still oblique relative to the ring axis, is deformed, to become flat as at 14, the outer rim diameter thereof is increased. outer rim diameter thereof is increased, whilst the internal rim diameter at the ring 80 bore 15 is maintained constant by means of a mandrel, and the edges 10 and 11 come into contact with the inclined surface 7a.

A further construction of the outer rim of a ring of this kind is illustrated in Figure 85 3. A ring 16, which is shown in the non-fitted condition with side faces 17 inclined relatively to the axis of said ring, has peri-pheral teeth 18 formed thereon, which teeth have only a small cross-section in propor-90 tion to the total cross-section of the ring. The comparatively very small teeth 18 are easily deformed when they contact a groove surface such as is designated with the reference numeral 7 and 7a respectively in Figures 1 and 2, without substantial radial forces being exerted on the ring 16 itself. Thus the projections exert only slight pressure on the bottom of the groove, the pressure however being sufficient to fix the ring in position and hance to prevent undesirable turning of the retaining ring. The amount of contact required for this purpose is kept so small that the parts which carry the mount-ing groove e.g. the races of needle bearings, 105 do not become deformed.

The axial retaining rings can also be used for the inner races of needle bearings in similar manner to that described with reference to Figures 1 to 3.

Figure 4 shows such an inner race 19 of a needle bearing, with race-way 20, a groove 21 in the said race-way carrying a retaining ring which comes into contact radially with the bottom of the groove only 115 around edge 22.

A similar ring 23 is shown in Figure 5 on a slightly larger scale, before it is pressed into the groove.

Before being deformed, the ring is brought into the plans of the groove 24, and is then so deformed by appropriate means that its aide retaining faces are dis-posed at approximately right angles to the axis of the needle bearing race. In so doing, 12st rim edge 25 first comes into contact with the bottom of the groove and provides the requisite contact for preventing rotation of the ring 23, without exerting any great pressure on the bottom of the groove 24 and 130

thus on the race itself.
WHAT WE CLAIM IS:

I. In, or for use in a needle roller bearing of the kind specified, the combination of an axial retaining ring and grooved race having relative dimensions such that an inner or outer rim of the ring will pross against the bottom of the groove with

inner or outer rim of the rim will pross against the bottom of the groove, with sufficient radial pressure to prevent rotation 10 of the ring in the groove, when the ring is fully spread into the groove, said rim being shaped so that the aforesaid pressure is transmitted by a surface area of the rim which is only a small fraction of the overall tim surface area and is insufficient to induce 15 run surface area and is insufficient to induce

deformation of the grooved par 2. A combination as claimed in Claim wherein the said rim is so constructed that when the ring is spread or pressed into
20 position only the leading edge of said rim,
in the direction in which the ring was in-

serted, bears against the bottom of the groove,

3. A combination as claimed in Claim
25 I, wherein the rim has a preferably central
peripheral recess, the arrangement being

such that when the ring is spread or pressed into position only the leading and rear edges of said rim will bear against the bottom of said groove.

A combination as claimed in Claim 1, wherein tooth-like projections are formed on said rim so that when the ring is spread or present into position within the groove said projections alone bear against the 35 bottom of the groove.

5. A needle roller bearing having in combination, a grooved race and an axial

combination, a grooved race and an axial retaining ring substantially as hereinbefore described and shown in Figures 1 or 2 or 40 4 and 5 of the accompanying drawings.

6. For use in a needle roller bearing having a grooved race, an axial retaining ring substantially as hereinbefore described with reference to Figs. 1 or 2 or 3 or Figs. 45 4 and 5 of the accompanying drawings. 4 and 5 of the accompanying drawings.

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Agents for the Applicants.

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